

Design of floating wind turbine system with DeepLines Wind

GDR-EMR : Journée Eolien Offshore



PRINCIPIA

Principia – Main activities

Offshore engineering activity



Expertise & rules justification



R&D to basic and detailed design

Expertise & analysis

Summary

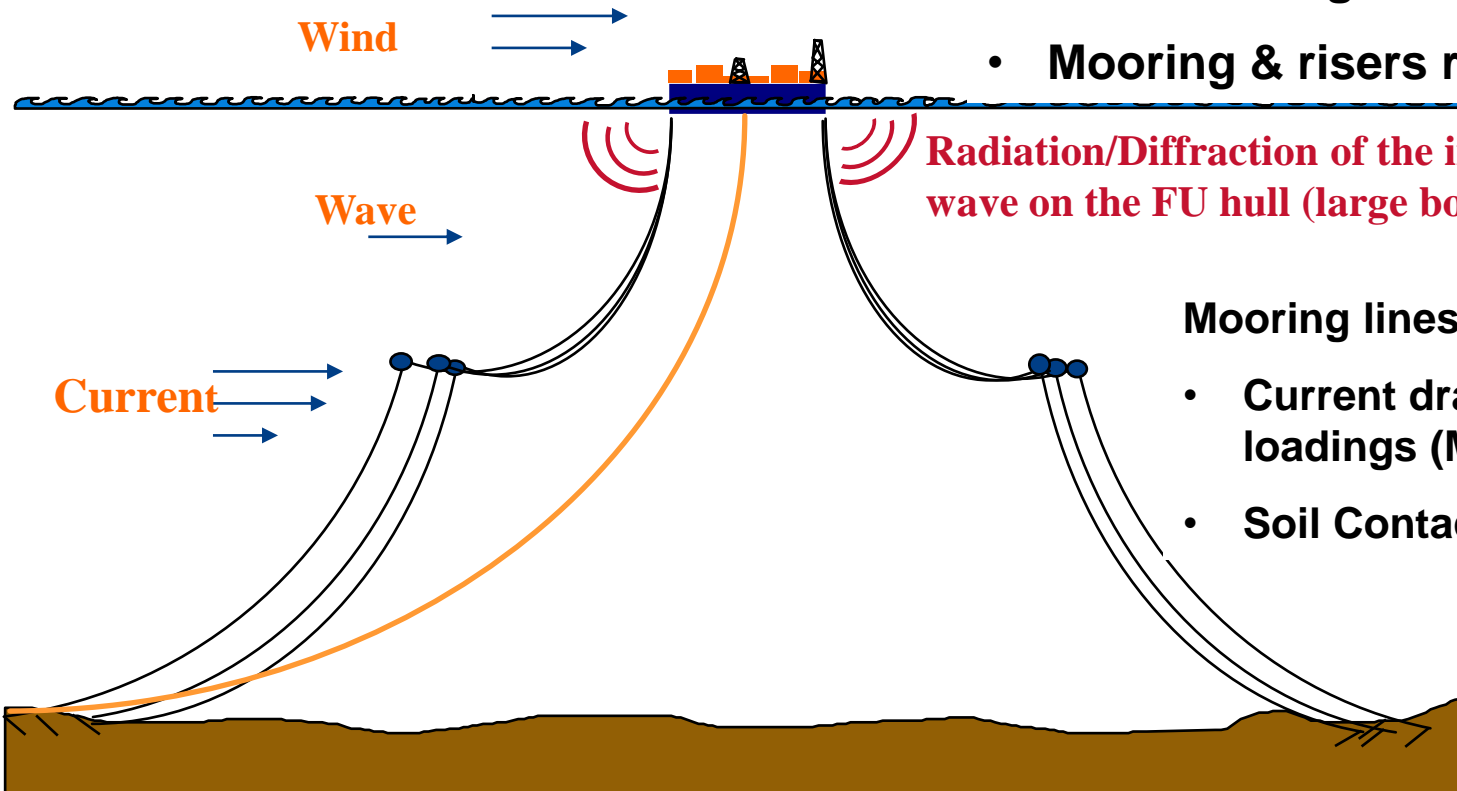
◆ **DeepLines Wind (in partnership with )**

◆ **Validation**

◆ **Design Considerations**

Floating system

◆ DeepLines



Floater side:

- Wind, current drag loads
- Wave loadings 1st and 2nd order
- Mooring & risers restoring forces

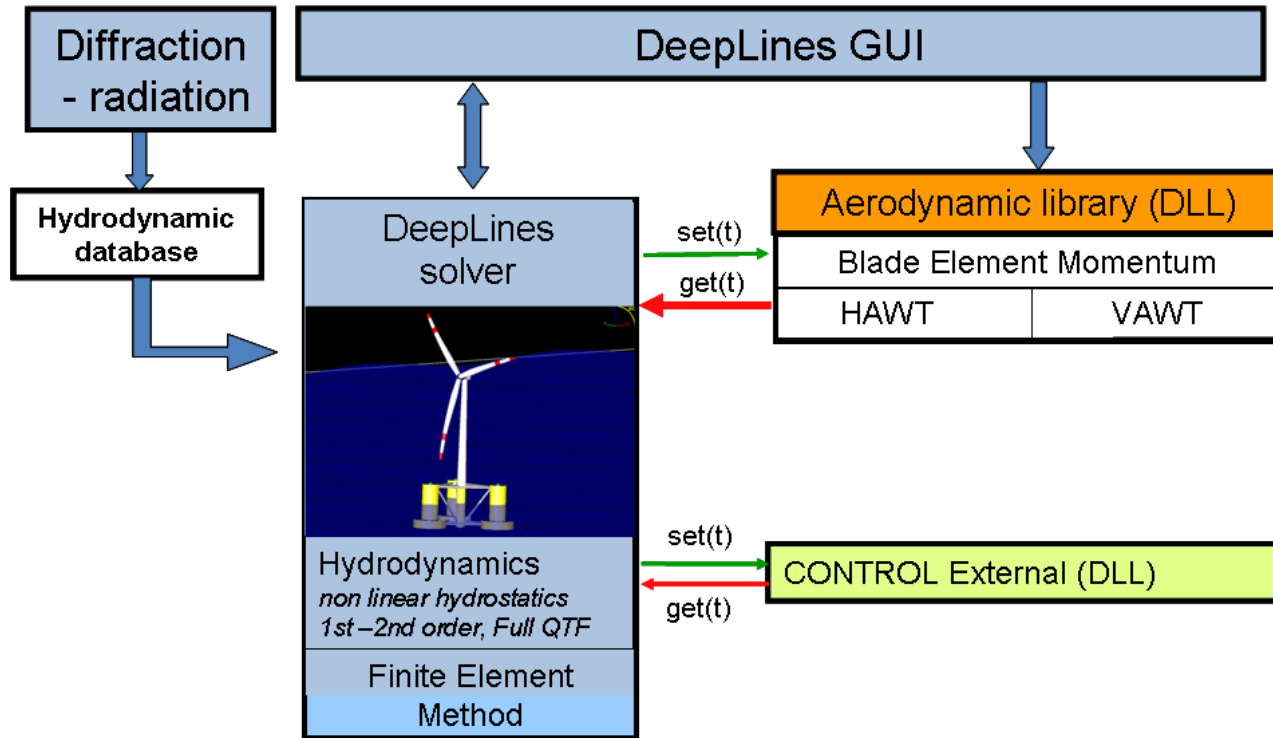
Radiation/Diffraction of the incident wave on the FU hull (large body)

Mooring lines & risers side:

- Current drag loads + wave loadings (Morison)
- Soil Contact

DeepLines Wind

Wind turbine modeling software based on DeepLines solver



New elements

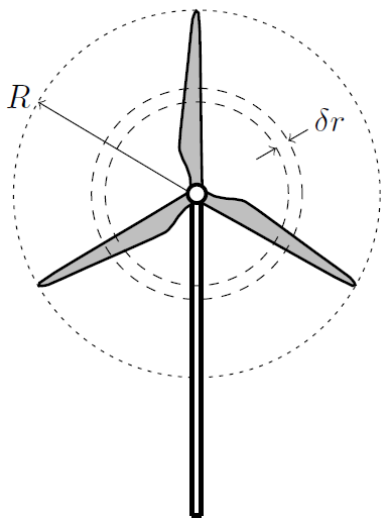
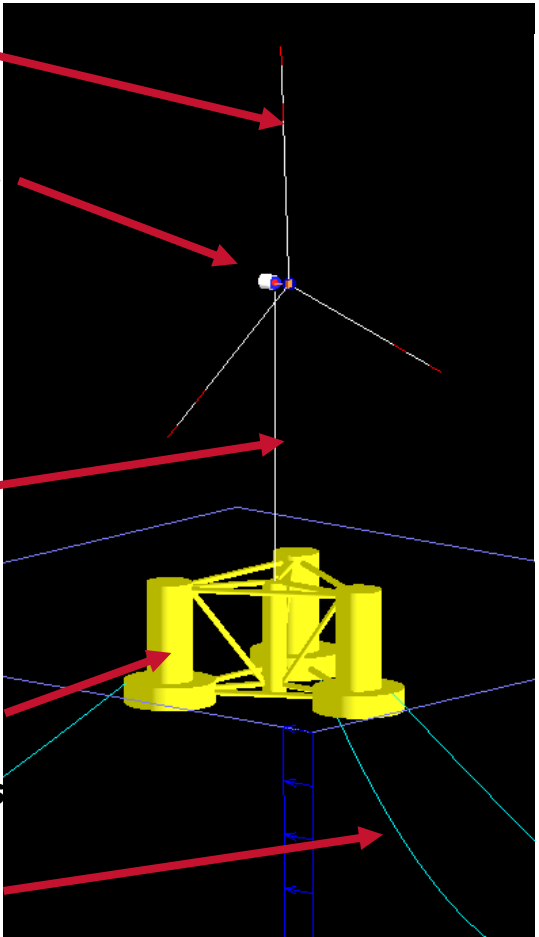
Blades :
beam elements

**Flexible nacelle /
tower connection :**
flexjoint

Tower :
beam elements

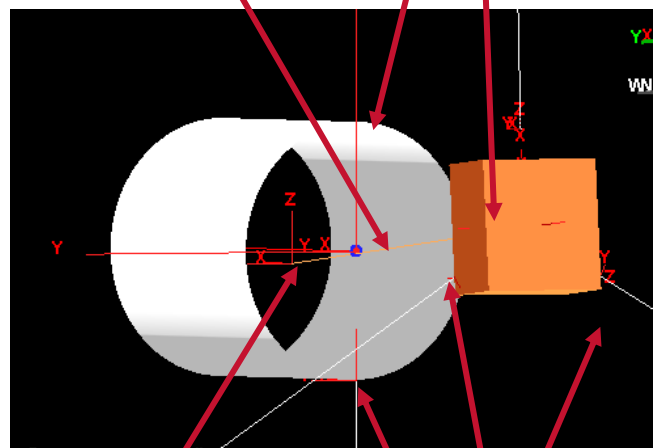
Floater :
**rigid floater
or beam elements**

Mooring lines



Hub and Nacelle :
rigid bodies

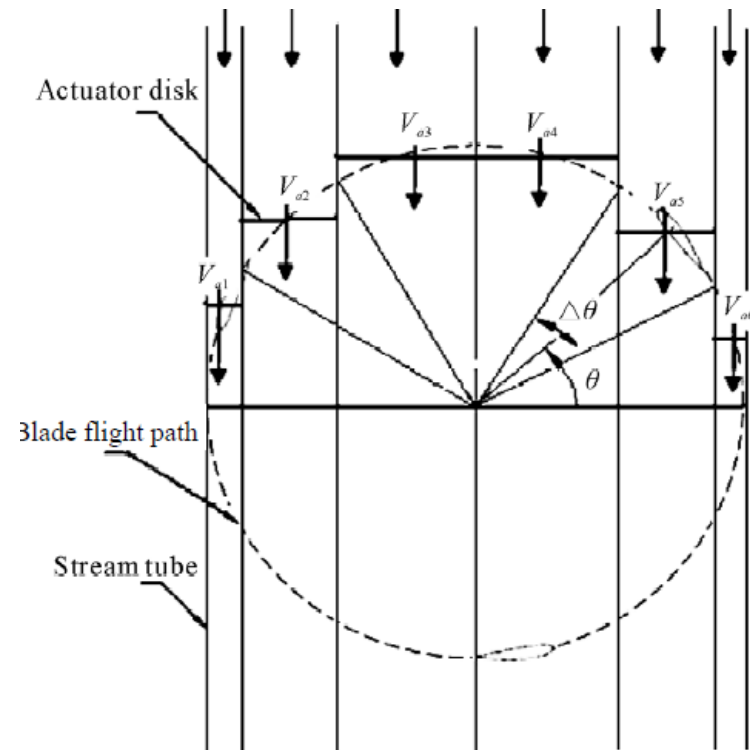
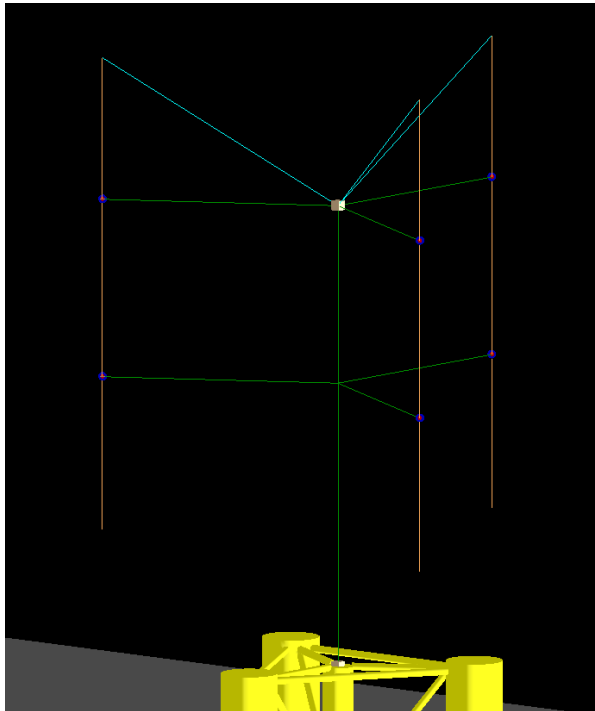
Low speed shaft :
beam elements



Generator :
node control

**Pitch/yaw
actuators :**
node control

- ◆ **Blades, struts and tower: beam elements**
- ◆ **Control**
- ◆ **Specific aerodynamic models**



FEM: Lagrangian Formulation

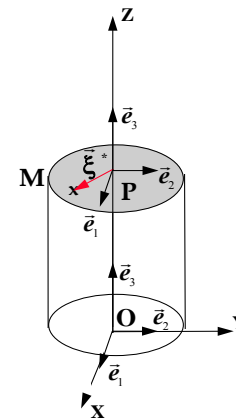
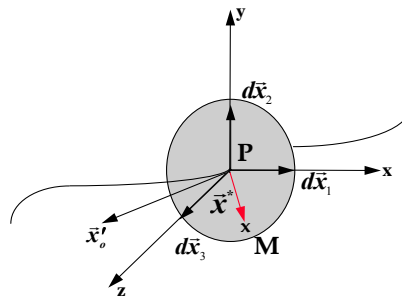
◆ Defines the local reference at this node: $\vec{d}_{xi} = \mathbf{R}_x(\vec{\theta}_x)\vec{e}_i$

◆ Let the rotation vector be $\vec{\theta}_x = \begin{pmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{pmatrix}$ and , the Rodrigues' formula is written as:

$$\mathbf{R}_x(\vec{\theta}_x) = I_d + \frac{\sin \theta}{\theta} \begin{pmatrix} 0 & -\theta_3 & \theta_2 \\ \theta_3 & 0 & -\theta_1 \\ -\theta_2 & \theta_1 & 0 \end{pmatrix} + \frac{1 - \cos \theta}{\theta^2} \begin{pmatrix} \theta_2^2 + \theta_3^2 & \theta_1\theta_2 & \theta_1\theta_3 \\ \theta_2\theta_1 & \theta_1^2 + \theta_3^2 & \theta_2\theta_3 \\ \theta_3\theta_1 & \theta_3\theta_2 & \theta_1^2 + \theta_2^2 \end{pmatrix}$$

$$\Omega_x \quad \begin{aligned} O\vec{P} &= \vec{x}_s(s) \\ O\vec{M} &= \vec{x}_s(s) + \vec{x}^* = \vec{x}_s(s) + \mathbf{R}(\vec{\theta}_x)\vec{\xi}^* \end{aligned}$$

$$\Omega_\xi \quad \begin{aligned} O\vec{P} &= s\vec{e}_3 \\ O\vec{M} &= s\vec{e}_3 + \vec{\xi}^* \end{aligned}$$



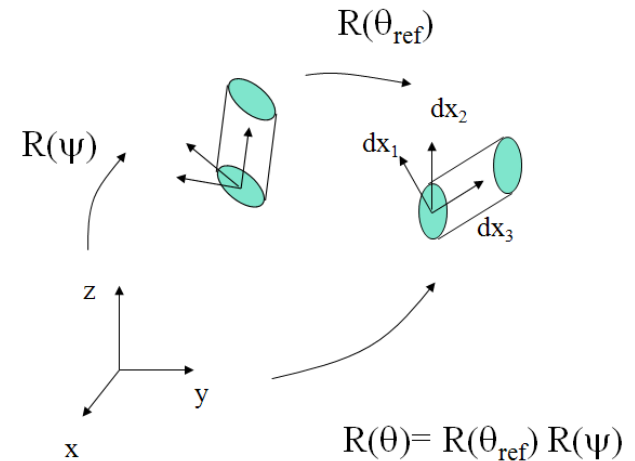
Infinite rotation handling : 2 options

Option 1

Updated Lagrangian Formulation

- The reference is the previous step (static or time)
- The unknowns are the variation from the reference position

Problem : adaptation of the code



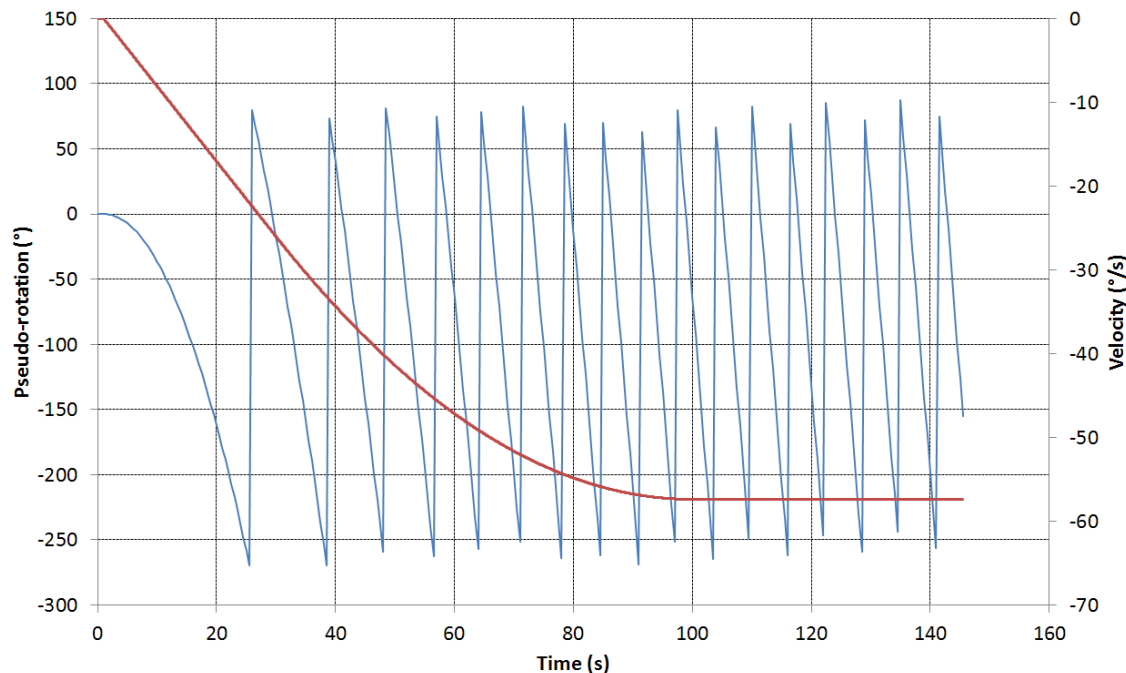
Option 2

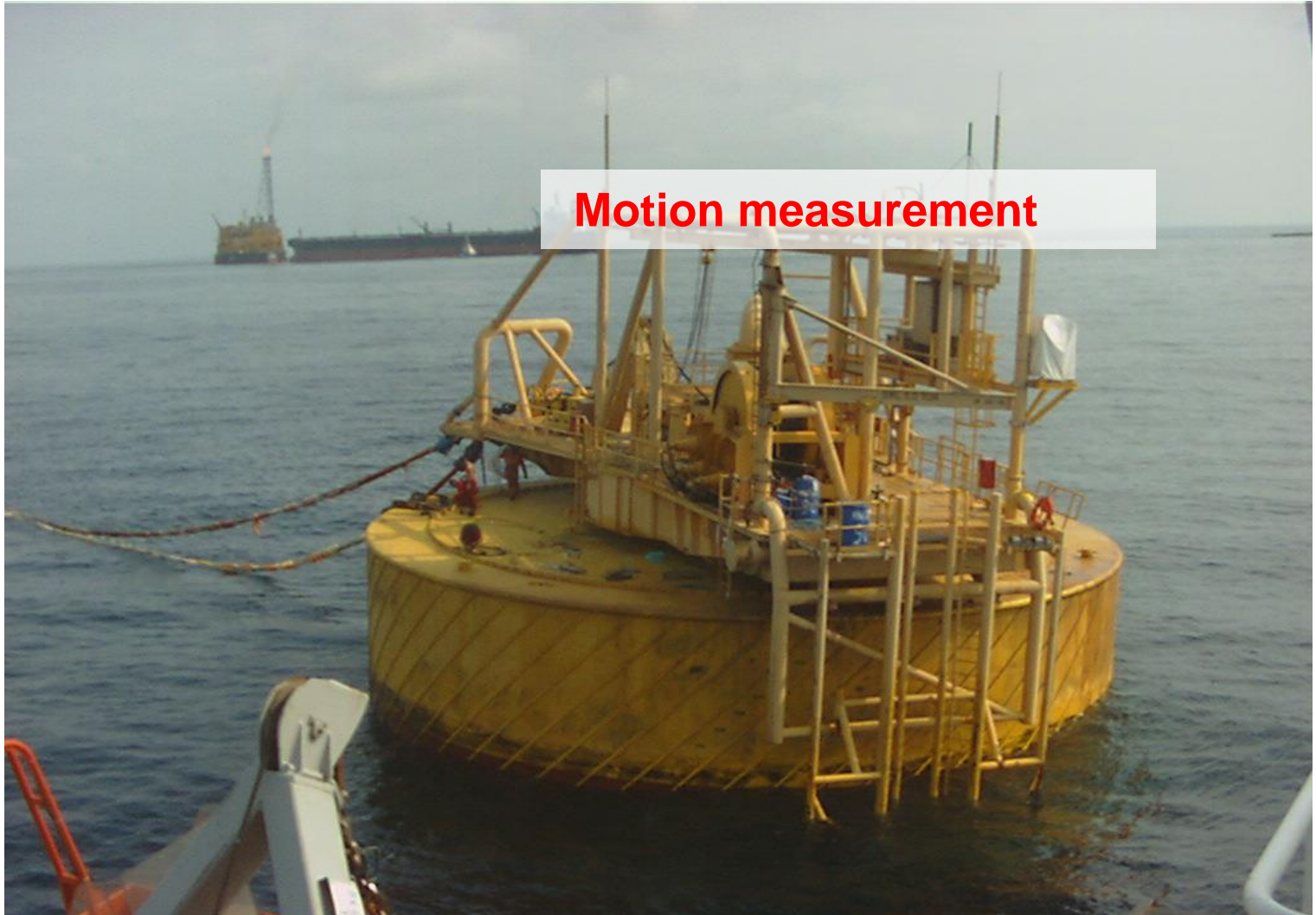
- ◆ Apply a transformation when necessary at the entrance of the step
- ◆ Proposed by Cardona, A., Gerardin, M., 1988
- ◆ Criteria on the norm of the pseudo-rotation vector to stay below 2π
- ◆ Introduce an updated pseudo-rotation vector:

$$\vec{\theta}_x^* = \left(1 - \frac{2\pi}{\|\vec{\theta}_x\|}\right) \vec{\theta}_x$$

Pseudo-rotation vector

- ◆ Pseudo-rotation vector becomes discontinuous with time at a given node
- ◆ The quantities rotational velocity and acceleration are continuous
- ◆ Numerical time marching scheme does not damp the results (constant rotational speed with constant moment)

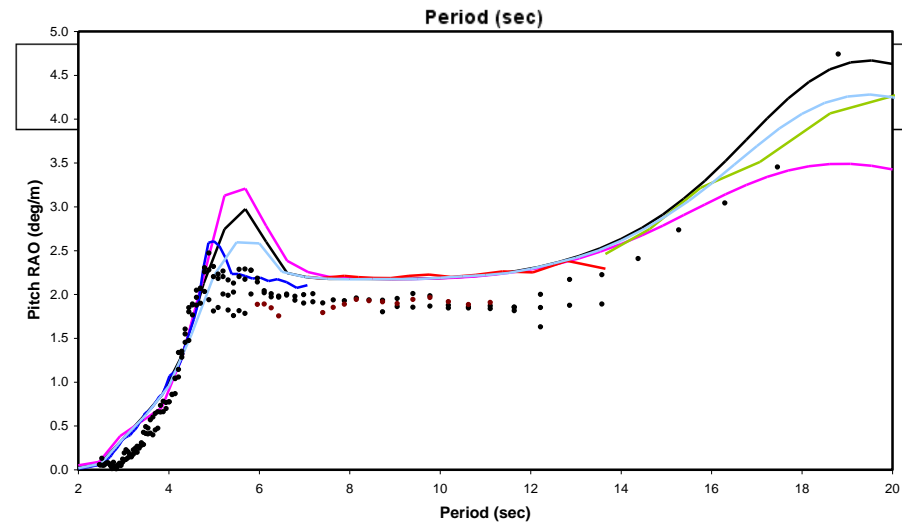
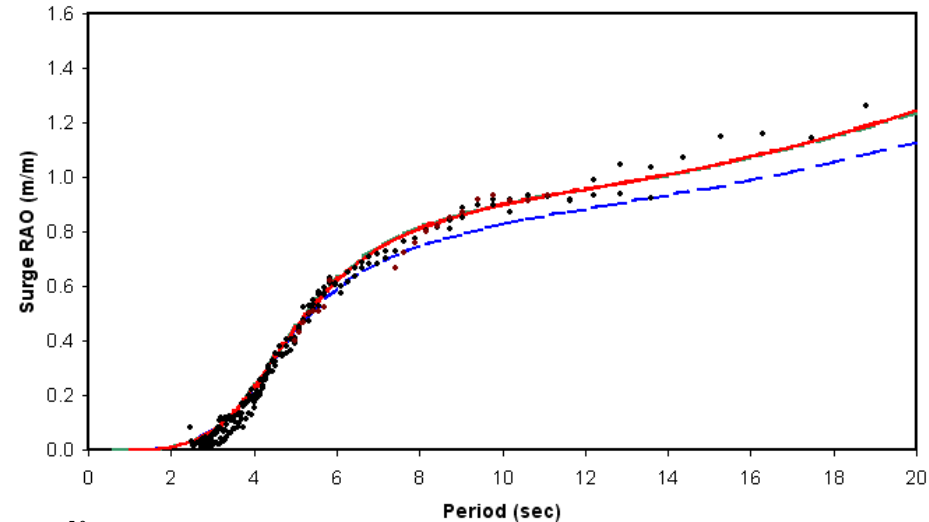
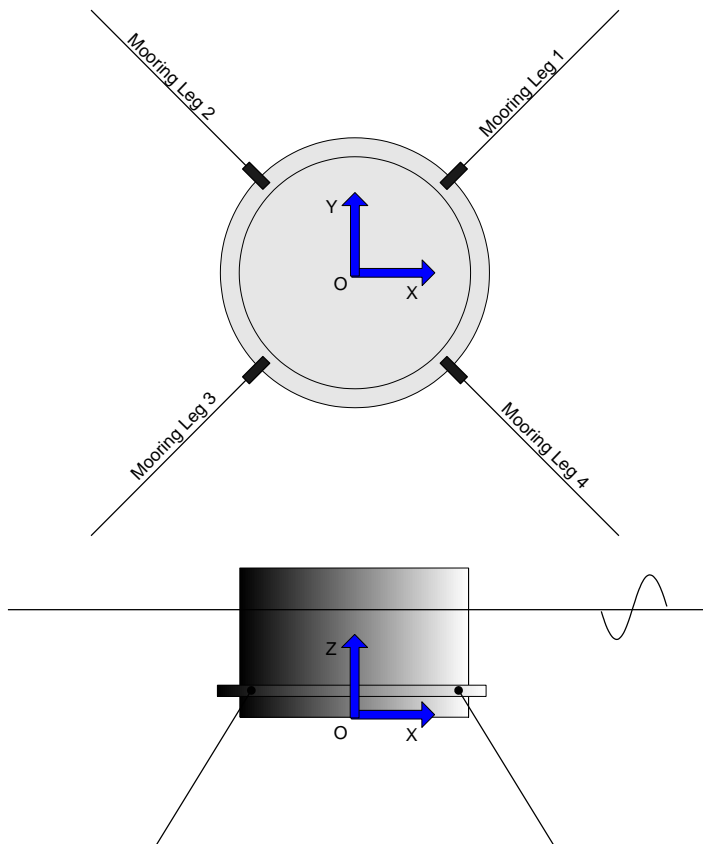




Motion measurement

Hydrodynamic/Mechanical coupling

Validation with respect to model test : anchored buoy coupled motion (2007)

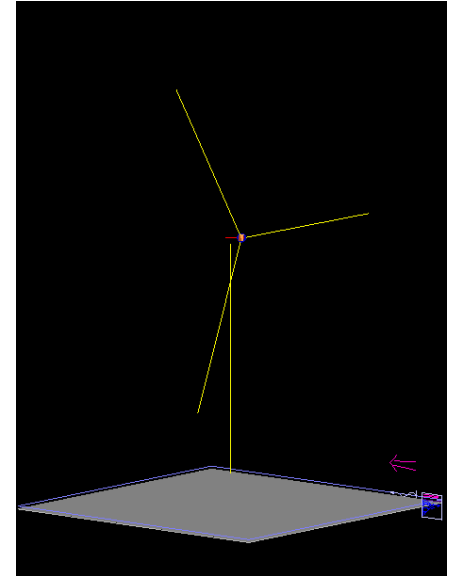


• model test
— freq-domain Reg 1m
— freq-domain Irreg Tp=4.5s
— time-domain Reg 1m
— freq-domain Irreg Tp=8s
— freq-domain Irreg Tp=15s
— freq-domain Reg 2m

Validation on OC3 cases Comparisons with FAST

◆ OC3 wind turbine with rigid foundation

- ◆ Blades : 3 * 34 beam elements - damping 0.5%
- ◆ Shaft : 1 beam element – damping 1%
- ◆ Tower : 20 beam elements – damping 0.3%
- ◆ Nacelle : rigid body (RB) with inertia matrix
- ◆ Hub : RB with inertia matrix
- ◆ Flexible tower / nacelle coupling

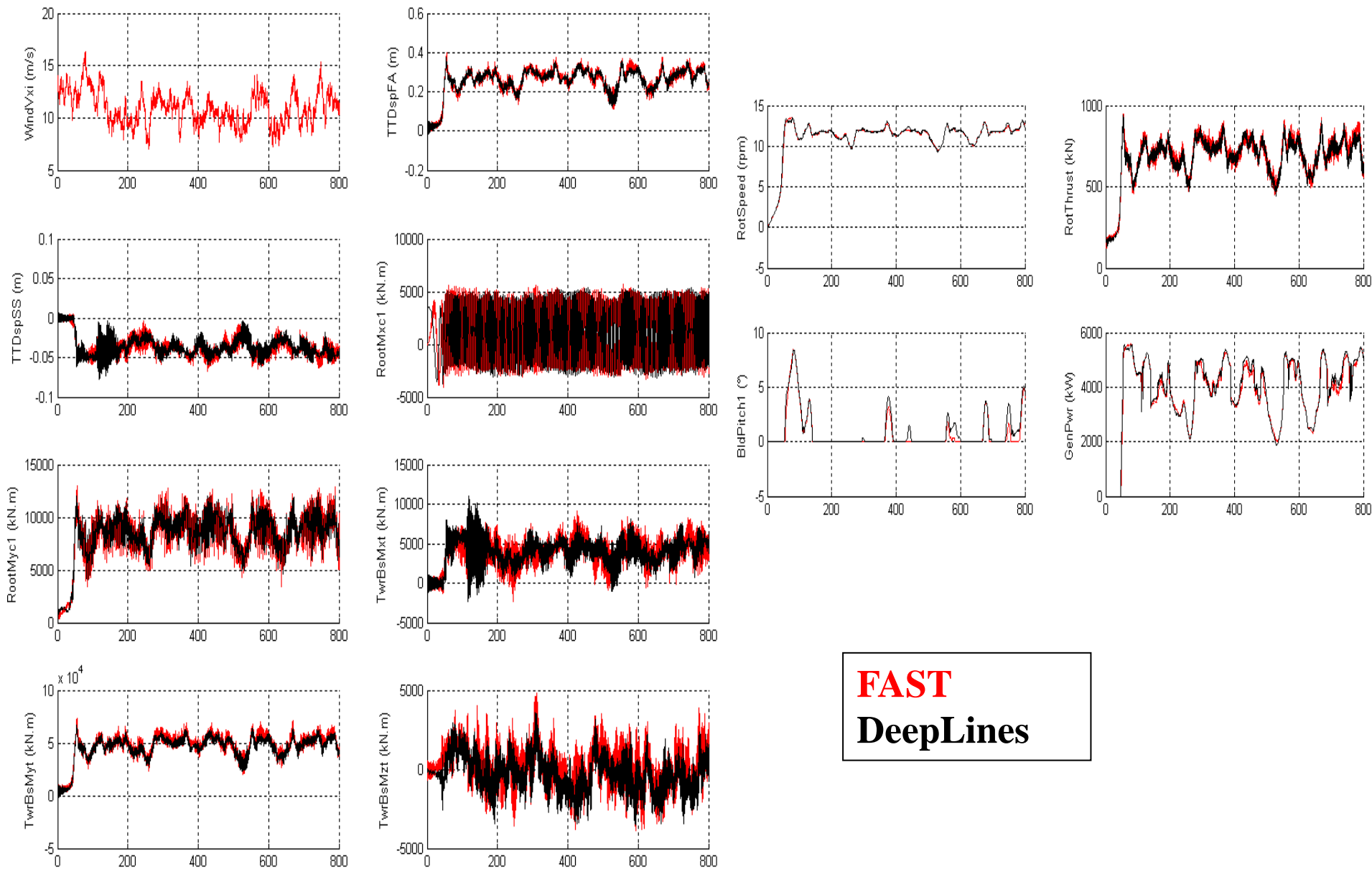


◆ Load cases

- ◆ HH1: constant wind speed: 2-25 m/s – Vertical gradient 0.14
- ◆ HH2: Linear ramp 5-50 m/s - yawed conditions @ 20°
- ◆ HH3: Extreme operating gust 25m/s
- ◆ NTM1: 3D turbulent wind - 11.2 m/s










Variable ID	Description	Unit
'TTDspFA'	fore aft nacelle displacement	meter
'TTDspSS'	side to side nacelle displacement	meter
'RootMxc1'	blade root moment due to the rotor in plane forces	kN.m
'RootMyc1'	blade root moment due to the rotor out of plane forces	kN.m
'TwrBsMxt'	tower Base Moment due to the side to side rotor force	kN.m
'TwrBsMyt'	tower Base Moment due to the rotor fore-aft force	kN.m
'TwrBsMzt'	tower Base Moment in yaw	kN.m
'RotSpeed'	rotor rotational speed	rpm
'RotThrust'	rotor thrust	kN
'BldPitch1'	blade pitch angle	°
'GenPwr'	generated power	kW

NTM1 3D turbulent wind@11.2 m/s – time series

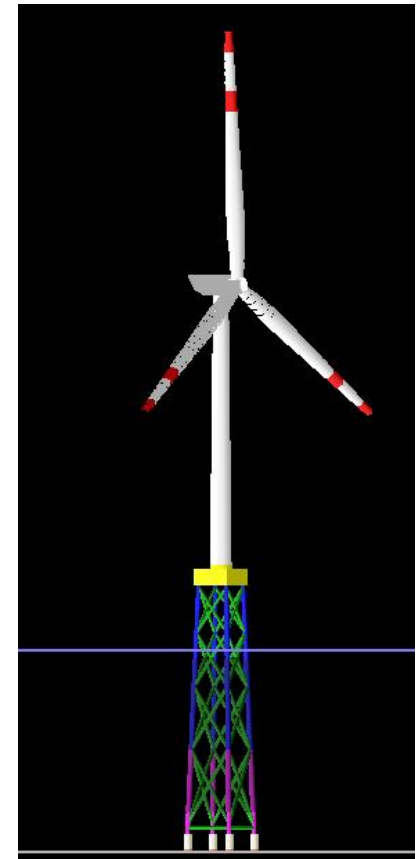


FAST
DeepLines

OC4 phase 1 - LC 5.6

Fraunhofer IWES/ADCoS-Offshore	
FEDEM/WindPower	
POSTECH/Bladed V3.85	
SWE/Flex5-Poseidon	
CeSOS-NTNU/USFOS-vpOne	
NTUA/GAST	
LUH/Flex5-Poseidon-WaveLoads2	
ABS/FAST-ANSYS	
Risø DTU/HAWC2	

PRINCIPIA/ Deeplines WT 

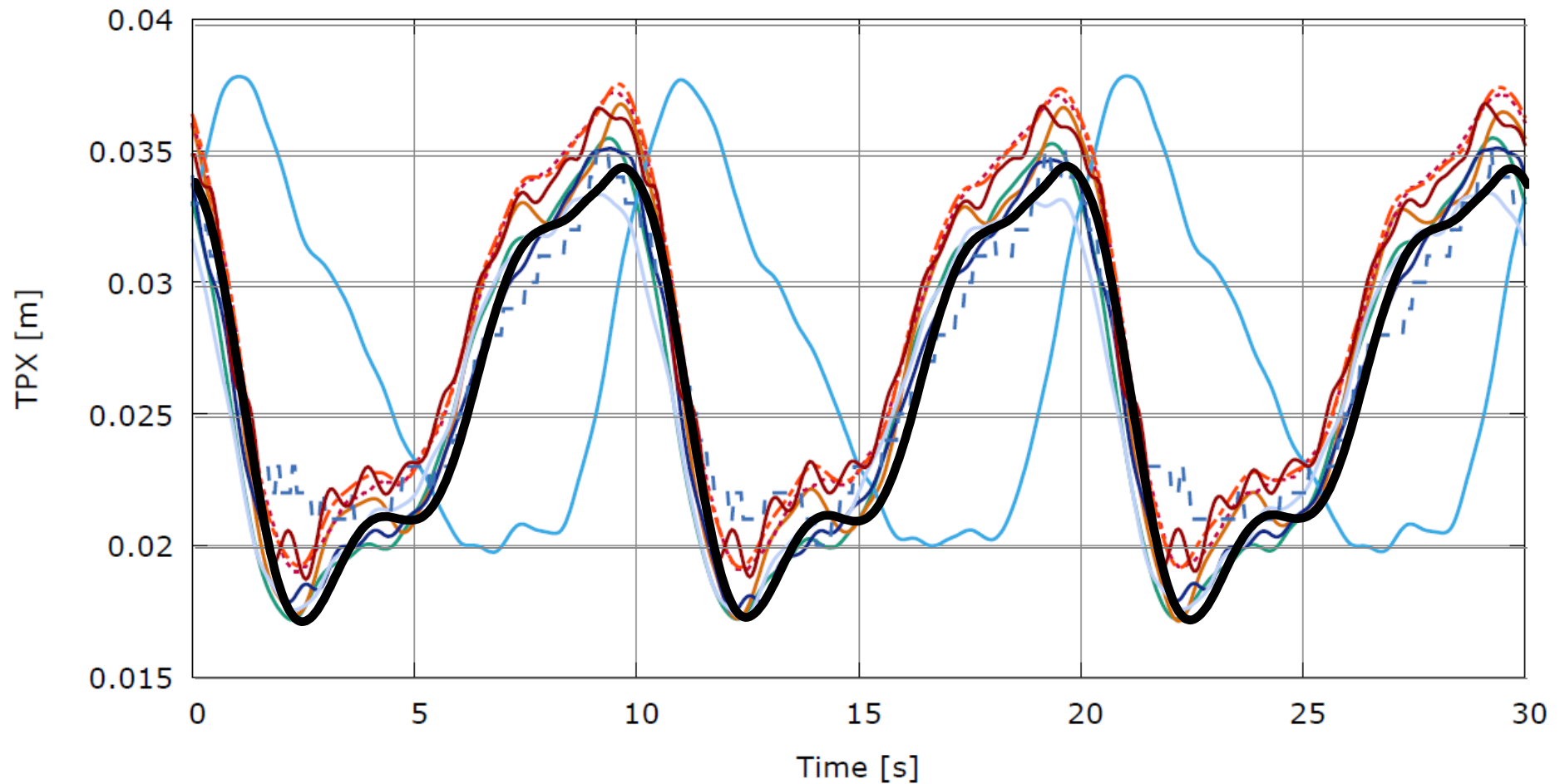


Load Case	Enabled DOF	Wind Conditions	Wave Conditions	Analysis Type	Initial Conditions
5.6	All, Rotor speed and blade pitch via controller	Steady, uniform, no shear: $V_{hub} = 8 \text{ m/s}$	Regular stream function (Dean, 9th): $H = 8\text{m}$, $T = 10\text{s}$	Periodic time-series solution	$\Omega = 9\text{rpm}$ $\Phi = 0\text{deg}$ $\Theta = 0\text{deg}$ wave simulation starts from crest at $x = 0, y = 0$ (global system)

Structure Deflection

Transition Piece Fore-Aft Deflection

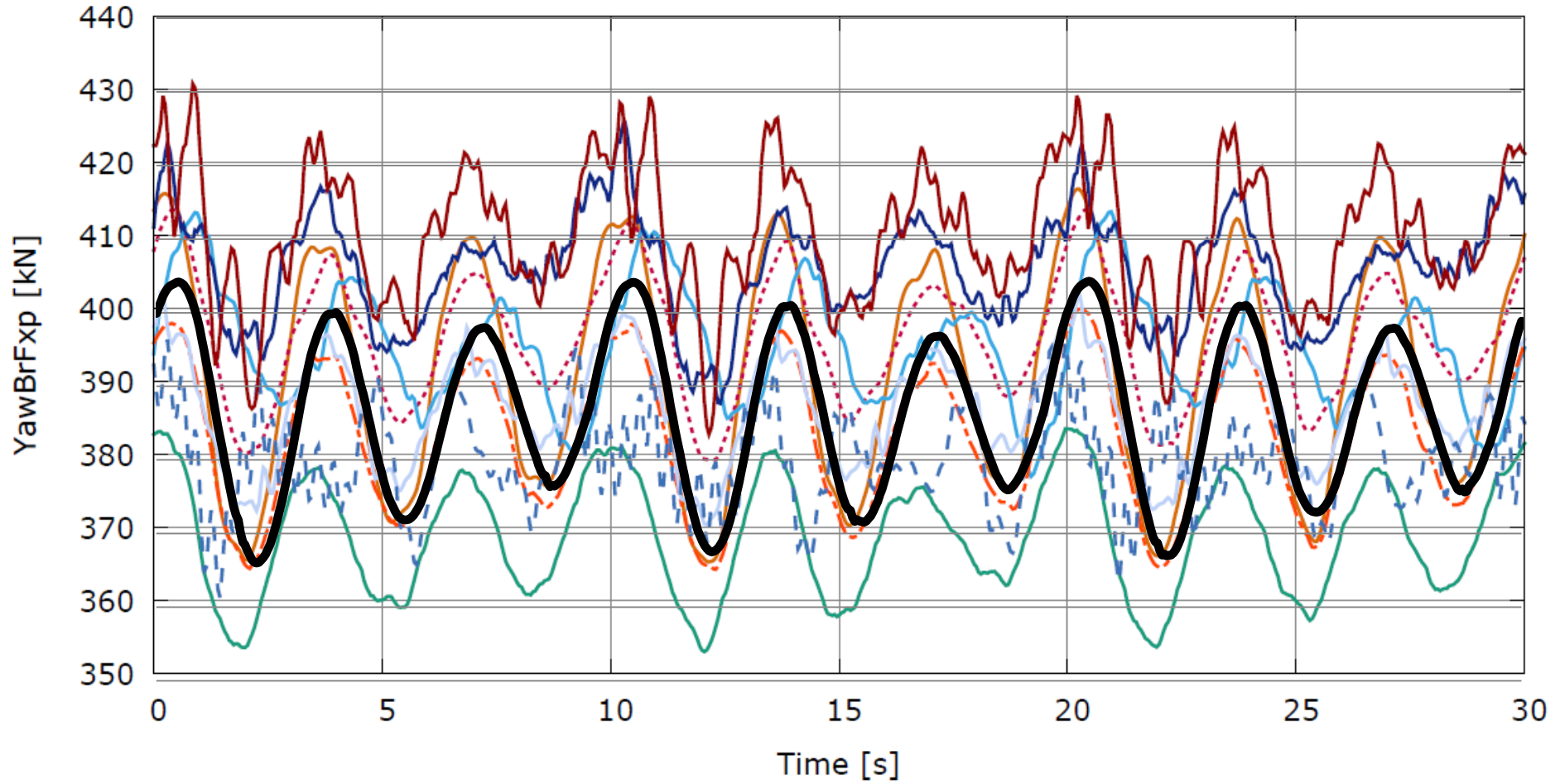
35 OC4 - Phase 1 - LC 5.6 Rev4 - 10.02.2012 - 08:29



Aero loads on Tower

Tower Top Fore-Aft Shear Force

29 OC4 - Phase 1 - LC 5.6 Rev4 - 10.02.2012 - 08:29



OC4 phase 2

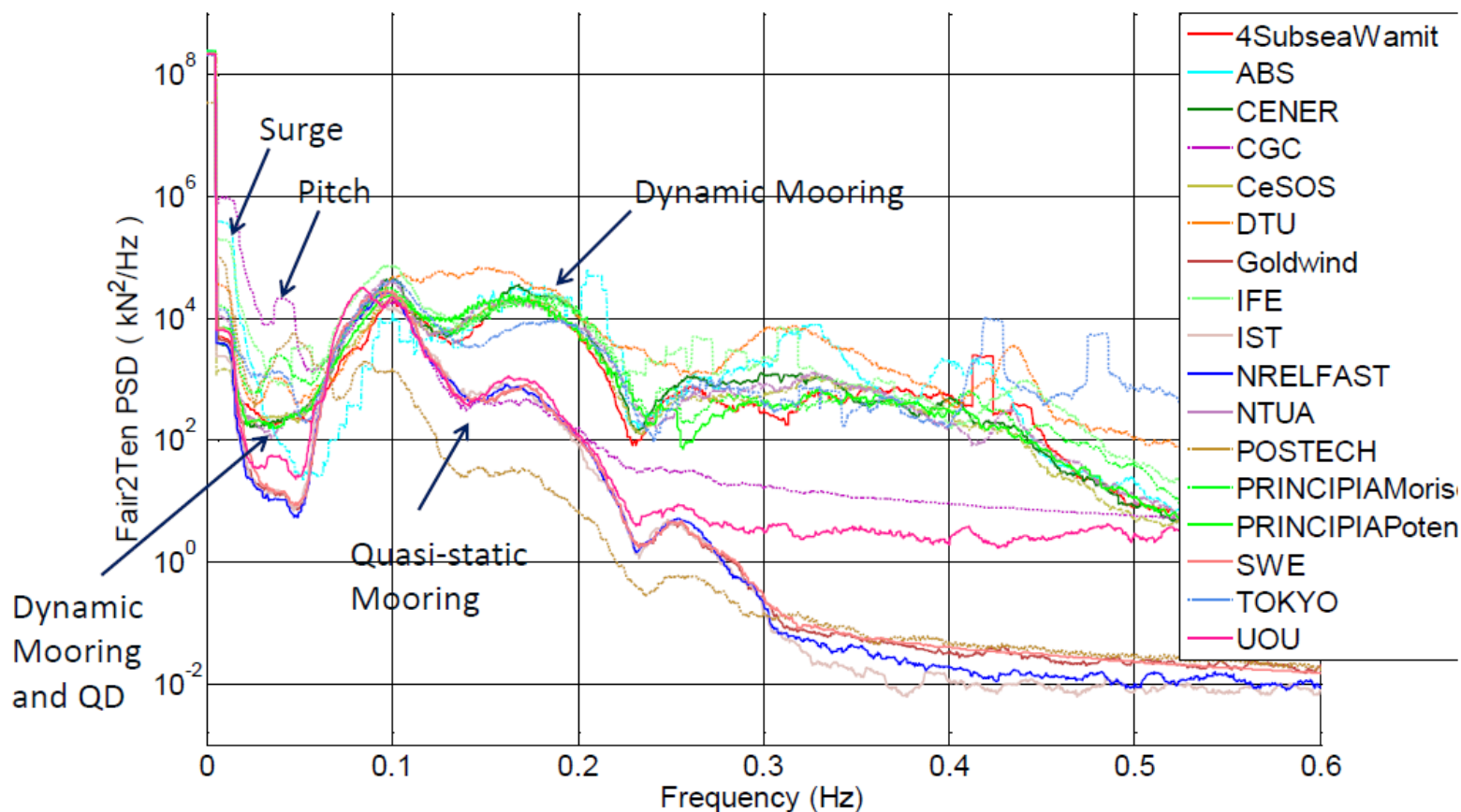


Table 4: Load-Case Set 2, Inverted Pendulum

Load Case	Description	Enabled DOFs	Sim. Length	Wind Condition	Wave/Current Condition	
2	Inverted pendulum (flexible support structure; rigid nacelle, drivetrain, and rotor; generator locked)					
2.1	Deterministic waves	Support structure	60 s	No air	Regular Airy: $H_s = 6$ m, $T = 10$ s	Time series
2.2	Stochastic waves	Support structure	60 min	No air	Irregular Airy: $H_s = 6$ m, $T_p = 10$ s, $\gamma=2.87$, JONSWAP spectrum	Time-series-generated PDFs and PSDs
2.3	Current only	Support structure	N/A	No air	Surface = 0.5 m/s, $1/7^{\text{th}}$ power law decrease with depth	Quasi-steady equilibrium position and loads
2.4	Current and deterministic waves	Support structure	60 s	No air	Regular Airy: $H_s = 6$ m, $T = 10$ s waves; Current at surface = 0.5 m/s, $1/7^{\text{th}}$ power law decrease with depth	Time series
2.5	50-year extreme wave	Support structure	60 min	No air	Irregular Airy: $H_s = 15.0$ m, $T_p = 19.2$, $\gamma=1.05$, JONSWAP spectrum	Time-series-generated PDFs and PSDs
2.6	RAO estimation, no wind	Support structure	60 min	No air	White noise, $H_s = 6$ m	Time-series-generated "effective RAOs"

Code
OrcaFlex
OrcaFlex
OrcaFlex
CHARM3D + FAST
OPASS + FAST
Simo+Riflex+Aerody
Bladed 4.3
WAMSIM
HAWC2
Bladed 4.3
FAST
3DFLOAT
FAST
SWT
hydro-GAST
FAST
FAST + OrcaFlex
GH Bladed
DeepLinesWT
DeepLinesWT
SIMPACK
CAST
UOU + FAST
Wavec2Wire

LC 2.2 – Irregular Wave – Mooring Response



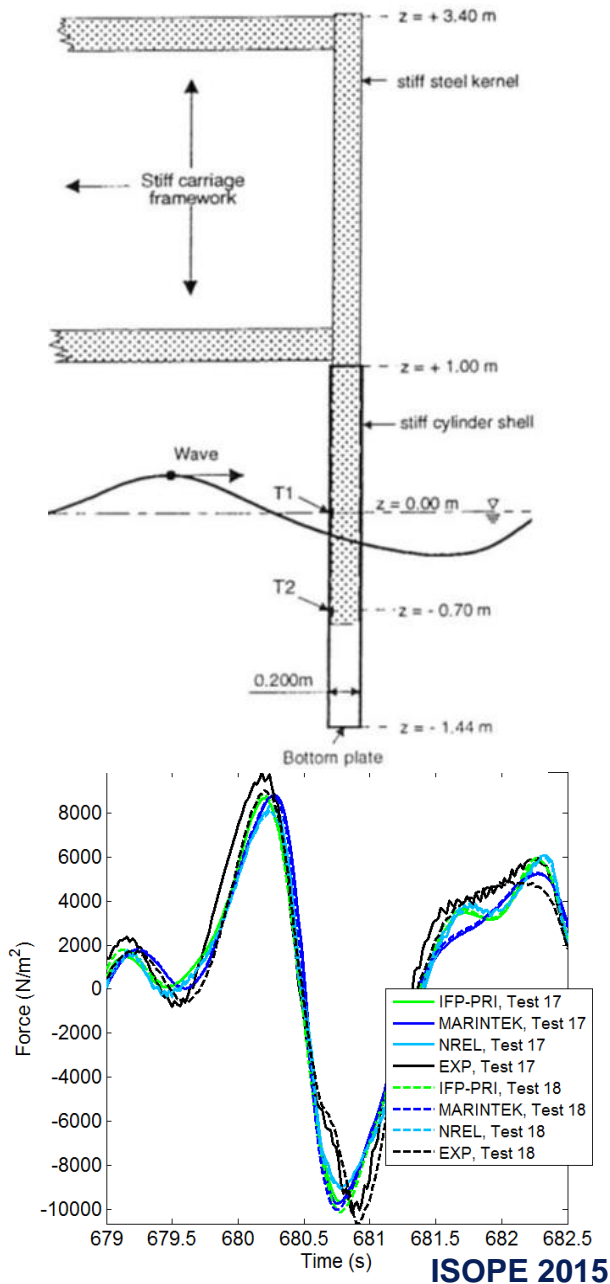
- Larger response for dynamic mooring models – excitation of mooring natural frequencies?
- Below wave frequency, Morison-based models show more response at platform frequencies
- At pitch frequency, see separation into three groups: QS+QD, Dyn+QD, Dyn+Mor

OC5 phase 1 and 2

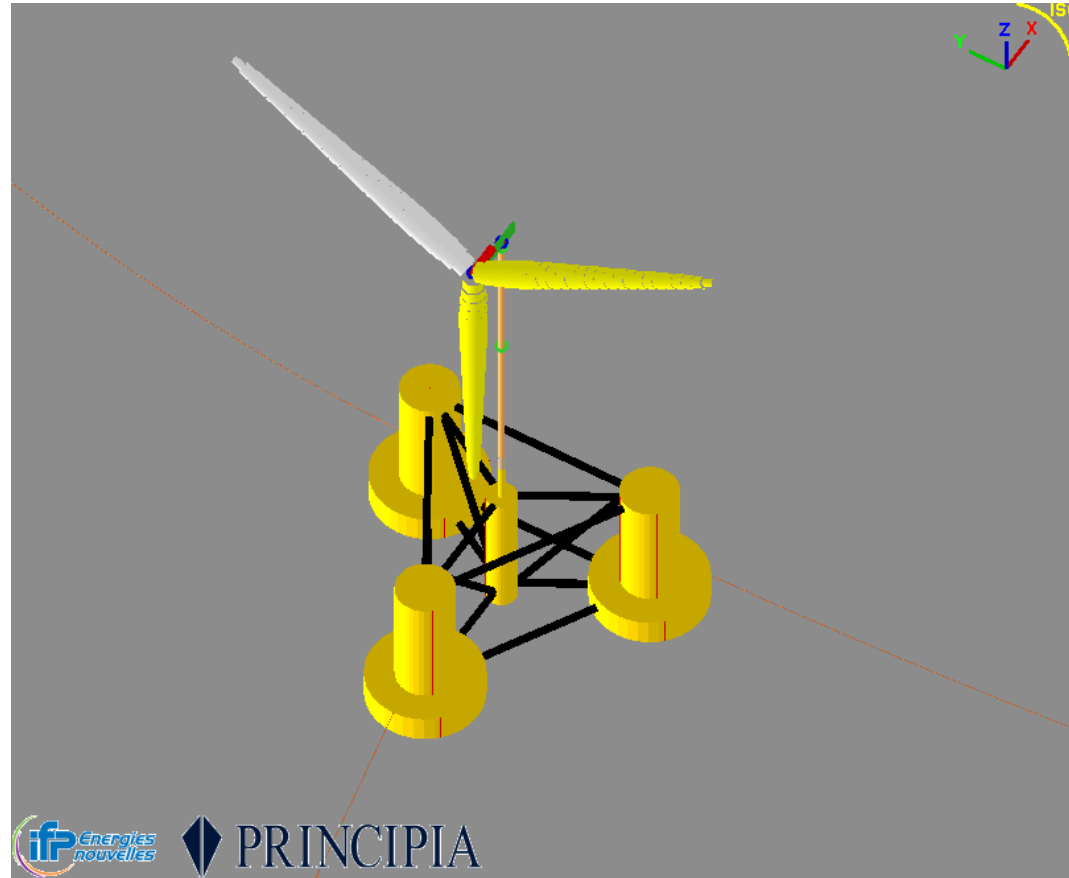
Model test:

Wave Forces

Floating windturbine



ISOPE 2015



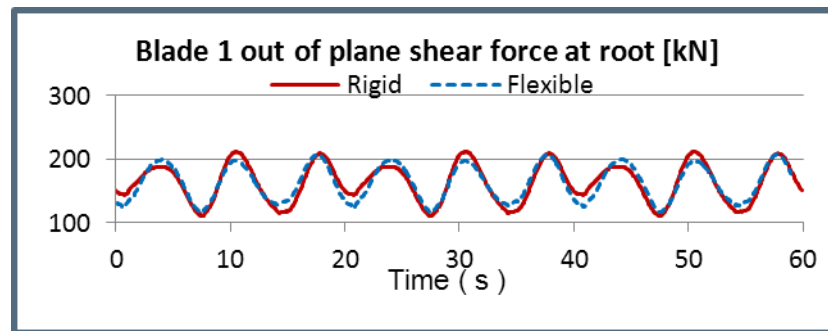
Statoil Hywind

- ◆ **Beam model: mooring lines, floater, tower, blades, shaft,...**
- ◆ **Floater beam modeling calibrated with a potential model**
- ◆ **Time domain simulation of the whole coupled system (hydro/aerodynamic loadings and turbine controller)**
- ◆ **Large number of ULS, FLS & ALS analyses**
- ◆ **Verification of design criteria (acceleration, pitch, offset, angles, etc...)**
- ◆ **Mooring design (extreme and fatigue)**
- ◆ **Floater, Tower design (extreme and fatigue)**
- ◆ **Comparison and verification of the simulations with Company and turbine manufacturers' software.**

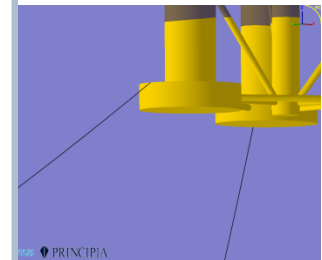
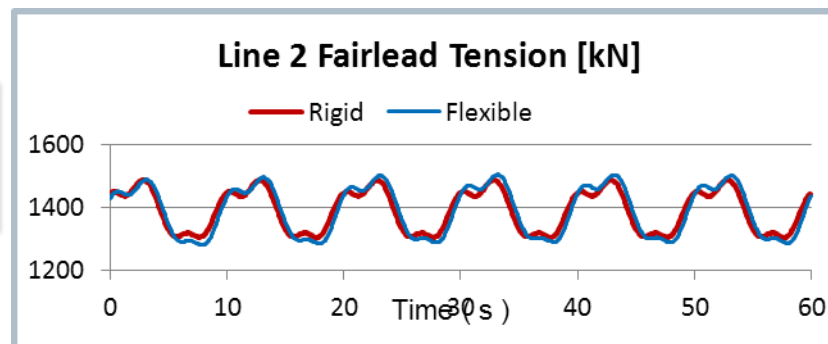


DeepWind Conference – Jan 2016

Low impact on blade loads

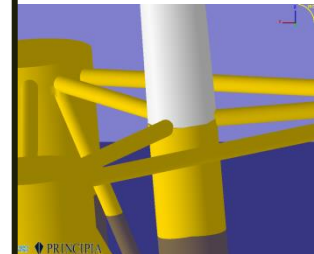
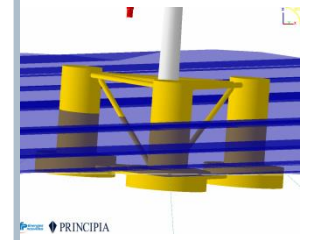
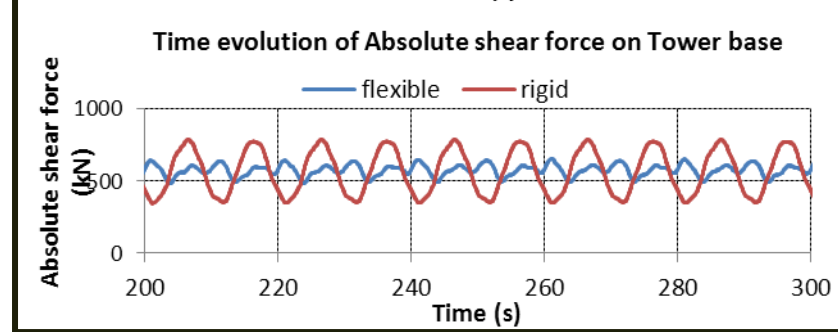
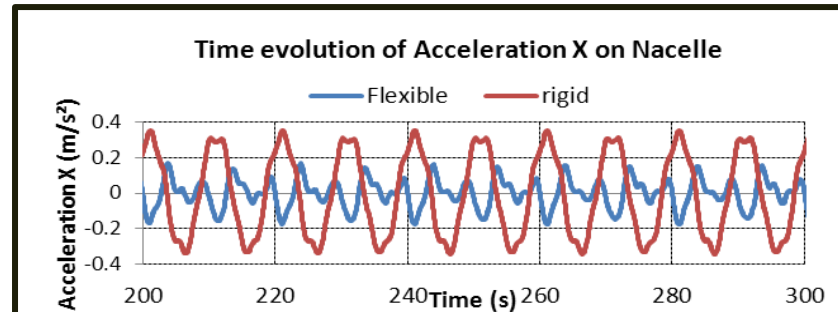
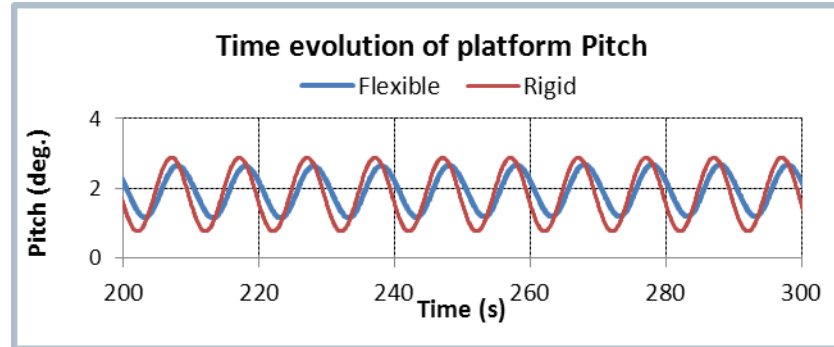


Low impact on Mooring lines



Low impact on platform motions (for usual wave loadings)

Considerable impact on Nacelle acceleration
Tower loads
platform loads



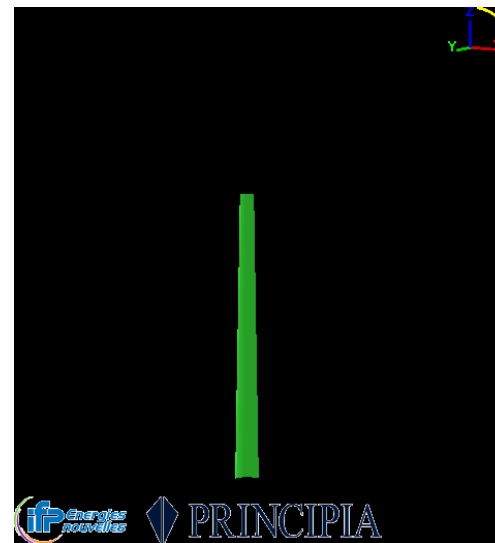
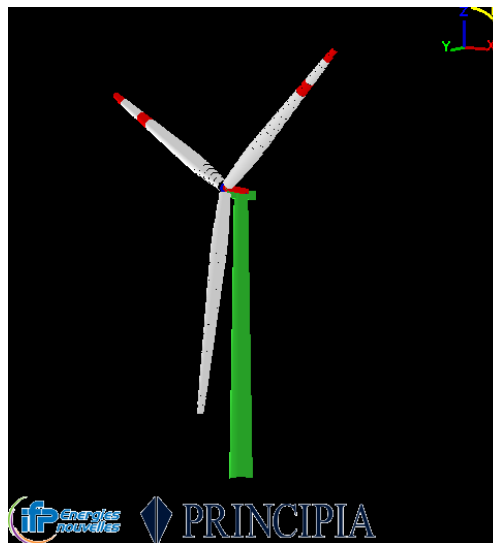
- ◆ **Bladed Interface**
- ◆ **Specificity of floating turbine**
- ◆ **IT issues**

Cross Validation

- ◆ **Aerodynamic / Controller**
 - ◆ Onshore cases
 - ◆ Offshore cases
- ◆ **Hydrodynamic**
 - ◆ Model Tests
 - ◆ Prototypes
- ◆ **Hydrodynamic simplified model**

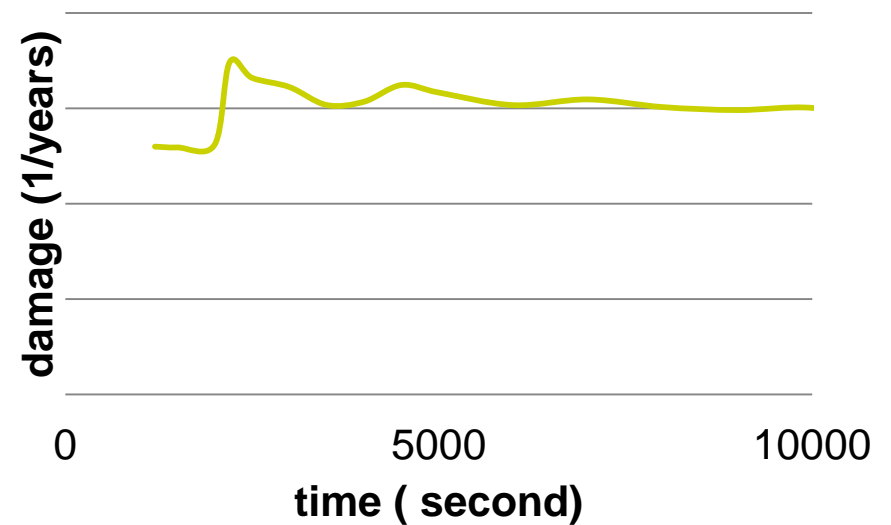
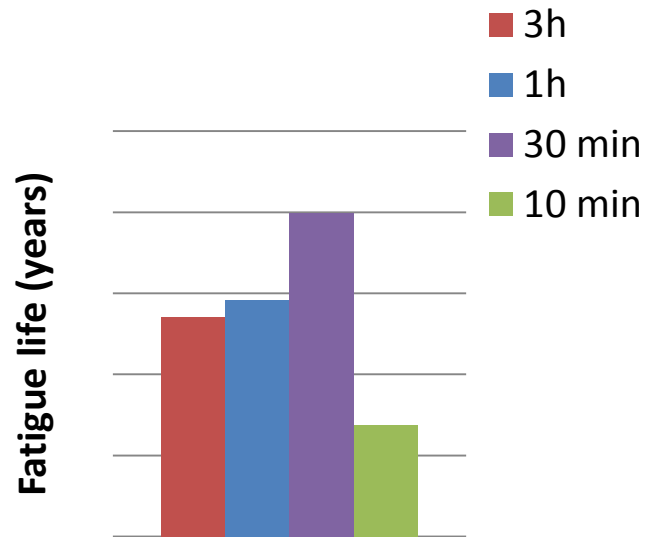
Simplified aerodynamic model

- ◆ Turbine / Cd / Constant load
- ◆ Extreme / Fatigue analysis



Load Cases

- ◆ Turbine vs Mooring / Structure design
- ◆ 10 min / 3 hours



◆ Floating wind turbine

- ◆ Interaction of two disciplines
- ◆ Two designs in one with their own focus
- ◆ Communication / Feedback

◆ Model

- ◆ Keep in mind the objective of your model
- ◆ Choose the proper simplifications